

# Wind-structure interaction simulations of ovalling vibrations in silo groups

J. Hillewaere<sup>a</sup>, J. Degroote<sup>b,\*</sup>, G. Lombaert<sup>a</sup>, J. Vierendeels<sup>b</sup>, G. Degrande<sup>a</sup>

<sup>a</sup>*Department of Civil Engineering, KU Leuven,  
Kasteelpark Arenberg 40, BE-3001 Heverlee, Belgium*

<sup>b</sup>*Department of Flow, Heat and Combustion Mechanics, Ghent University,  
Sint-Pietersnieuwstraat 41, BE-9000 Ghent, Belgium*

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## Abstract

During a storm in October 2002, wind-induced ovalling vibrations were observed on several empty silos of a closely spaced group of silos in the port of Antwerp (Belgium). In this paper, three-dimensional numerical simulations are used to investigate this complex case of wind-structure interaction. The computed amplitude of the ovalling vibrations of the silos is similar to that in the observations, indicating that the adopted modelling approach can be suitable for the analysis of new silo groups.

Both one-way and two-way simulations are presented, for a single silo and for the silo group. In the one-way simulations, the wind pressure is applied on the structure, disregarding the structural displacements in the wind flow simulation. By contrast, the two-way simulations also take into account the effect of the structural motion on the wind flow. For a single silo, the one-way and two-way simulations yield similar results. Conversely, for a silo in the group, the ovalling vibrations are significantly larger in the two-way simulations than in the one-way simulations. Consequently, aeroelastic effects and/or interactions between the wake-induced excitation and the vibration are present in the silo group for the investigated case.

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\*Corresponding author

*Email addresses:* jeroen.hillewaere@bwk.kuleuven.be (J. Hillewaere), joris.degroote@ugent.be (J. Degroote), geert.lombaert@bwk.kuleuven.be (G. Lombaert), jan.vierendeels@ugent.be (J. Vierendeels), geert.degrande@bwk.kuleuven.be (G. Degrande)

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Furthermore, it is shown that the aerodynamic loading and vibration amplitudes are considerably larger for silos in the group than for a single isolated silo.

*Keywords:* wind-structure interaction, aeroelastic instability, wind-induced vibration, ovalling, silo

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## 1. Introduction

Given the tendency to build ever taller, more slender and hence increasingly flexible structures, many present-day constructions are susceptible to wind-induced vibrations. Although mostly very small, these vibrations can become excessive and there are many examples of catastrophic structural failure due to wind-induced vibrations (Païdoussis et al., 2011; Belloli et al., 2011; Sheldard, 1967; Billah and Scanlan, 1991). For the design of flexible structures, it is therefore increasingly important to understand the mechanisms that are causing these vibrations and to provide engineers with proper methodologies and tools to investigate these phenomena.

In the present paper, the case of a group of 40 silos in the port of Antwerp (Belgium) is studied (figure 1). During a storm in 2002 wind-induced ovalling vibrations with an amplitude of approximately 0.1 m have been observed on several empty silos at the windward side of the group. This vibration is significant compared to the gaps of 0.3 m between the silos. As is typical for ovalling deformations, the cross section of the axisymmetrical structure deforms as a shell, without bending deformation with respect to the longitudinal axis of symmetry (Païdoussis and Wong, 1982).

The ovalling vibrations of the silo group show some similarity with the collapse of 3 cooling towers from a group of 8 at Ferrybridge (UK) during a gale in 1965 (Pope, 1994). However, these collapsed cooling towers were located at the leeward side of the group while the ovalling vibrations in the silo group occurred on the windward side. A major difference between the silo group and the cooling towers is the size of the gaps between the individual objects, respectively 5%



Figure 1: Southwest corner of the silo group in the port of Antwerp (Belgium) where ovalling vibrations were observed in October 2002.

25 of the diameter compared to approximately one diameter. This entails that the  
 26 flow blockage effect for the silo group is much higher than for the cooling towers.  
 27 Furthermore, the cooling towers are made of reinforced concrete as opposed to  
 28 aluminium for the silo group, resulting in lower damping for the latter. The  
 29 investigation of this incident with the cooling towers identified a tensile merid-  
 30 ional failure within the shell fabric as dominant initial mode of failure. For the  
 31 particular wind direction during the gale, the mean meridional force increased  
 32 by 30% and the resonance force doubled, whereas the quasi-static fluctuating  
 33 force remained more or less the same compared to an isolated tower (Pope,  
 34 1994).

35 Because of the complexity of this wind-structure interaction (WSI) problem,  
 36 a simplified phenomenological model or experimental methods are not suitable  
 37 to investigate the ovalling vibrations. Other aeroelastic applications, e.g. flutter  
 38 of bridge decks or galloping of cables can be modelled with a two degrees of  
 39 freedom system in a wind tunnel (Païdoussis et al., 2011). However, a scale  
 40 model of a silo structure that has the appropriate flexibility is not easily con-  
 41 structed. Furthermore, in situ measurements of the structural response under  
 42 normal wind loading by Dooms et al. (2006) could not decisively pinpoint what  
 43 mechanism is causing the ovalling vibrations.

44 By contrast, the versatility of numerical techniques to simultaneously incor-  
 45 porate complex wind flow details and structural flexibility, even for problems  
 46 with complex geometries, is a great advantage. Procedures are available to  
 47 couple a numerical model for the wind flow, i.e. computational fluid dynamics  
 48 (CFD), and the structure, e.g. finite element (FE) models. Hence, this WSI  
 49 problem can be studied numerically. Numerical studies on the flutter of bridges  
 50 have been performed by Mannini et al. (2011a) and Šarkić et al. (2012), for  
 51 example. Furthermore, the motion of light-weight membrane structures under  
 52 wind load is analyzed numerically by Hojjat et al. (2010) and Michalski et al.  
 53 (2011). In addition, the deflection of wind turbine blades during operation is  
 54 calculated by Bazilevs et al. (2011). However, the behaviour of silos and other  
 55 cylinders under wind loading is still often investigated without taking the struc-  
 56 tural displacement into account in the flow calculation (Uernatsu et al., 2015;  
 57 Gorski et al., 2015; Zhao et al., 2013, 2014).

58 In this paper, WSI simulations are performed to investigate the wind-induced  
 59 overvibrations of individual silos mounted in a group arrangement. Com-  
 60 pared to our previous work related to this silo group (Hillewaere et al., 2012),  
 61 a significantly different methodology is proposed. Our previous work only con-  
 62 tained CFD simulations, with rigid structures. In this work, flexible structures  
 63 are considered, with and without including the influence of their motion on the  
 64 wind flow. The model of the silo group was 2D in our previous work and has  
 65 been extended to 3D in this paper, which among other things entails including  
 66 the atmospheric boundary layer. Consequently, only frequencies and circumfer-  
 67 ential mode shapes could be calculated in our previous work, without changes  
 68 along the axis of the silo, and no magnitudes of the vibrations. In addition, the  
 69 turbulence modelling has been improved to resolve a fraction of the turbulent  
 70 fluctuations. Finally, the results in our previous work overestimated the excita-  
 71 tion by the flow in the wake of the silo group because the simulations were 2D  
 72 which preserves large vortices and prevents their break up.

73 The goal of this paper is threefold. First, the necessity of taking into account  
 74 the structural displacement in the flow calculations for this aeroelastic problem

75 is analyzed. The differences between the results with and without including  
76 displacement in the flow will shed light on the mechanisms possibly causing  
77 the vibrations. Second, the effect of the group arrangement on the magnitude  
78 of the vibrations is determined by comparing by WSI simulations for a single  
79 silo with those for the entire closely spaced silo group. As mentioned above,  
80 the cooling towers at Ferrybridge experienced higher loading but they are more  
81 widely spaced so it is unclear whether that finding also applies to the silo group.  
82 Third, it is assessed whether the adopted modelling approach is suitable as  
83 an engineering tool for the analysis of new silo groups and possibly of similar  
84 structures to avoid the ovalling vibrations.

85 The paper is organized as follows. In section 2, details of the ovalling phe-  
86 nomenon are given and the present state of knowledge on wind-induced ovalling  
87 vibrations is briefly sketched. In section 3, the methodology for performing WSI  
88 simulations is explained in detail. The numerical models and techniques used for  
89 modelling both the wind flow and the silo structures are subsequently explained  
90 in section 4. In section 5, the results of the WSI simulations are described.  
91 Finally, the main conclusions of the study are given in section 6.

## 92 **2. Observation and physical interpretation of ovalling vibrations**

93 Wind-induced ovalling vibrations have been observed on several empty silos  
94 at the windward side of a group consisting of 40 silos in the port of Antwerp  
95 (Belgium) during a storm on 27 October 2002. Based on observations and an  
96 in-depth analysis of available low resolution video footage of the ovalling event,  
97 the amplitudes of the ovalling shell oscillations were estimated in the order of  
98 0.1 m. Displacements of this order of magnitude are reasonably large compared  
99 to the silo diameter  $D = 5.5$  m and the limited space of 0.3 m between adjacent  
100 silos. The largest vibrations were observed at the windward corner of the silo  
101 group (silo 1 in figure 2). Other silos at the windward side also showed ovalling  
102 but with smaller amplitude.

103 As shown in figure 3, the group of 8 by 5 thin-walled aluminium silos is

104 mounted at 16.66 m above ground level. Only the upper cylindrical part of the  
 105 silos is exposed to the incoming wind flow. The lower conical parts are embedded  
 106 in a rectangular building that is 49 m long and 31.6 m wide. Each silo is 25 m  
 107 high and is composed of aluminium sheets with a varying thickness decreasing  
 108 with height from 10.5 mm to 6 mm. Because the simulation of highly turbulent  
 109 wind flows around complex geometries is a challenging task, simplifications in  
 110 the computational model are required. Therefore pipes, cables, walking plat-  
 111 forms on top of the silos as well as the staircase next to the group will not be  
 112 considered in this analysis.

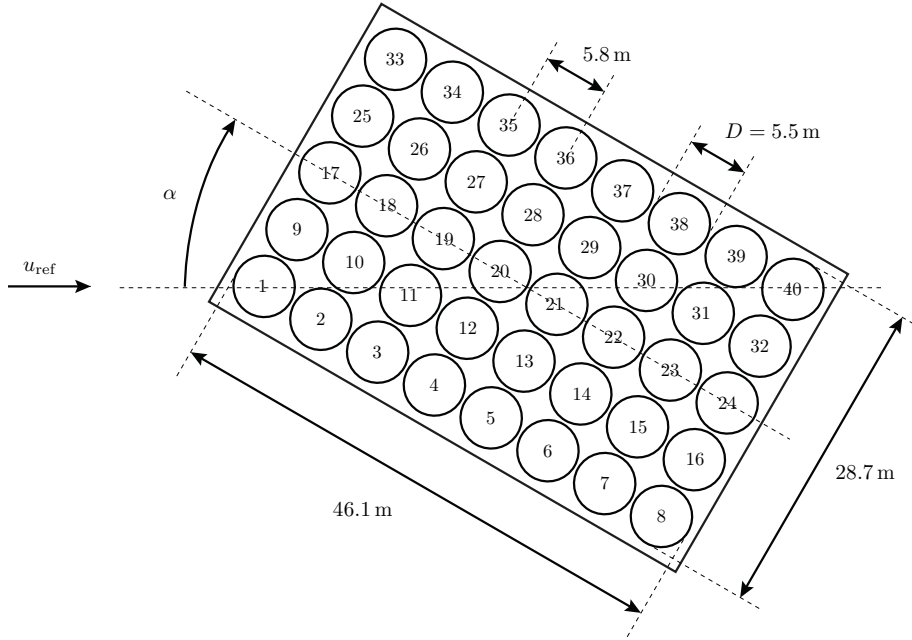


Figure 2: Plan view of the silo group with numbering of the individual silos. Normative dimensions are given as well as the definition of the angle of incidence  $\alpha$ .

113 No measurements are available of the wind field near the silo group at the  
 114 time the ovalling instabilities were observed. However, at a permanent meteo-  
 115 rological station in Deurne, about 7 kilometers east of the silo group, the hourly  
 116 average wind speed was monitored and ranged from 61 to 68 km/h with peaks up

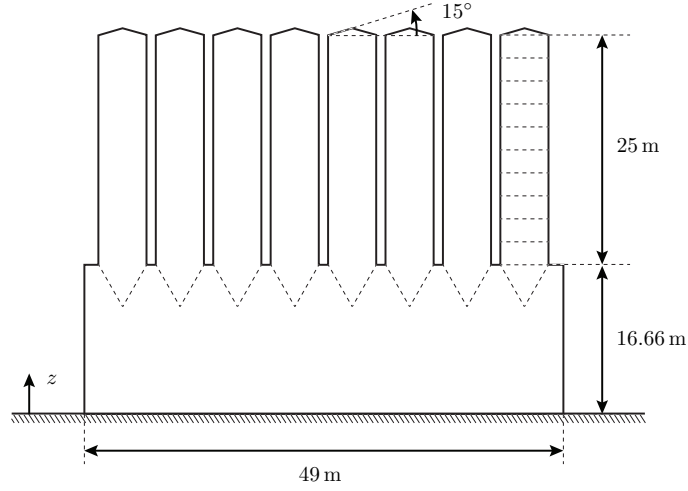


Figure 3: Lateral view of the silo group.

117 to 113 km/h. As indicated in figure 4, the silo group is located in flat surround-  
 118 ings near the river Scheldt and the main wind direction was west-southwest at  
 119 the time of ovalling (figure 4 and  $\alpha \approx 30^\circ$  in figure 2). With this information,  
 120 a loading condition can be created which is expected to be similar to the one  
 121 causing the observed ovalling in 2002 and sufficiently accurate to identify the  
 122 underlying physical mechanisms that caused the vibrations.

123 A distinction is generally made between three different excitation mecha-  
 124 nisms for flow-induced vibrations or more specifically wind-induced vibrations  
 125 (Blevins, 1990; Païdoussis et al., 2011; Weaver et al., 1993; Žukauskas et al.,  
 126 1988; Price et al., 1987): turbulence-induced vibrations, wake-induced vibra-  
 127 tions and fluidelastic instability (FEI). In the case of wind, the latter is also  
 128 called aeroelastic instability (AEI). Turbulence-induced excitation (or buffet-  
 129 ing) originates from fluctuations in the wind flow around the structure, e.g. from  
 130 natural turbulence in the wind flow attacking the structure. Wake-induced ex-  
 131 citation is due to periodic vortex shedding in the wake of the structure, possibly  
 132 with interaction (e.g. lock-in) between this wake and the structural motion. By  
 133 contrast, aeroelastic phenomena are self-excited and the fluctuating forces are



Figure 4: Location of the silo group near the river Scheldt in the port of Antwerp (Belgium). The mean wind direction during the storm of 27 October 2002 was west-southwest as indicated by the arrow. The shaded area denotes the flat region in front of the group.

134 due to structural motion. Also combinations of these phenomena occur.

### 135 3. Methodology

136 In this section, the one-way and two-way simulations are described. Comparing their results provides some insight into the excitation mechanisms of the  
 137 observed wind-induced vibrations. The calculation of the silo's deformation by  
 138 a structural solver and of the wind by a flow solver are discussed in more detail  
 139 in section 4. In the following paragraph, short notations for these solvers are  
 140 already defined.

142 The aerodynamic pressures and shear stresses  $\mathbf{P}(t_i)$  on the interface between  
 143 the structure and the wind flow — i.e. on the surface shared by the structural  
 144 calculation and the wind flow calculation — are determined in a wind flow  
 145 simulation for time step  $t_i$ . The structural solver  $\mathcal{S}$  calculates the structural  
 146 response of the silo due to these aerodynamic loads  $\mathbf{P}(t_i)$ :

$$\mathcal{S}[\mathbf{P}(t_i)] = \mathbf{U}(t_i). \quad (1)$$

147 The response of the entire structure is calculated but only the displacements  
 148 of the interface are passed back as  $\mathbf{U}(t_i)$ . The notation in this equation only



149 mentions information on the interface ( $\mathbf{U}$  and  $\mathbf{P}$ ). Obviously, also previous time  
 150 levels  $t_{i-1}, t_{i-2}, \dots$  and additional boundary conditions are taken into account  
 151 when calculating the solution, but these are not shown to keep the notation  
 152 clear.

153 Based on the interface displacements, the flow solver  $\mathcal{F}$  deforms the mesh in  
 154 the fluid domain close to the silo. Subsequently, the flow solver calculates the  
 155 wind in the entire fluid domain and returns the aerodynamic loading  $\mathbf{P}(t_i)$  on  
 156 the interface:

$$\mathcal{F}[\mathbf{U}(t_i)] = \mathbf{P}(t_i). \quad (2)$$

157 Similarly, the dependence on previous time levels, on additional boundary con-  
 158 ditions and on the remainder of the fluid domain are implicitly understood.

### 159 3.1. One-way approach

160 In the one-way approach, the aerodynamic loads are first computed for a rigid  
 161 structure, followed by calculation of the resulting structural response. Conse-  
 162 quently, wind-structure interaction is not considered in the one-way approach.  
 163 Because the structure is not moving in the aerodynamic load calculation, only  
 164 turbulence-induced vibrations and part of the wake-induced vibrations are ac-  
 165 counted for in this approach. Interactions between the wake-induced excitation  
 166 and the structural motion as well as aeroelastic effects are not captured.

167 The flow solver  $\mathcal{F}$  is used to determine the aerodynamic loading,  $\mathbf{P}(t)$ , acting  
 168 on the interface between structure and wind flow in every time step:  $\mathcal{F}[\mathbf{0}] =$   
 169  $\mathbf{P}(t_i)$ , with  $\mathbf{0}$  referring to the zero displacements of the rigid structure. As  
 170 illustrated in figure 5, the resulting time history  $\mathbf{P}(t)$  of aerodynamic loads  
 171 on the interface is subsequently applied as an external transient load on the  
 172 structure and the structural displacements are computed in the structural solver:  
 173  $\mathcal{S}[\mathbf{P}(t)] = \mathbf{U}(t)$ .

174 Due to an abrupt application of wind pressures on the undeformed silo struc-  
 175 tures, a long period of transitional effects would be observed in the structural  
 176 response. To avoid such a transitional regime, a static loading step is intro-  
 177 duced first:  $\mathbf{KU}_0 = \mathbf{P}_0$ . The pressures in the first time step of the aerodynamic

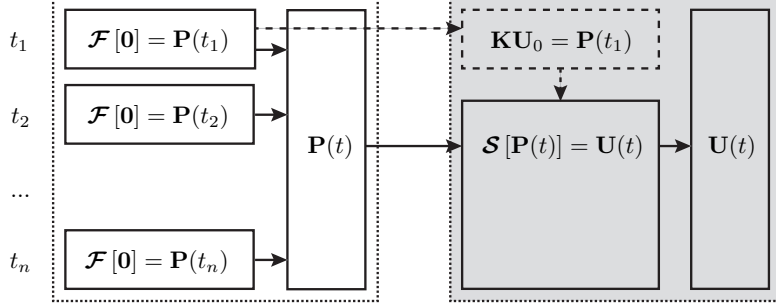


Figure 5: Schematic representation of the interaction between the flow solver  $\mathcal{F}$  (white) and the structural solver  $\mathcal{S}$  (grey) in the one-way partitioned simulation. The time-dependent aerodynamic loads  $\mathbf{P}(t)$  are calculated with zero structural displacement. Subsequently, they are transferred to the structural solver for the calculation of the time-dependent displacement  $\mathbf{U}(t)$ , starting with initial displacement  $\mathbf{U}_0$  from a steady calculation.

simulation  $\mathbf{P}_0 = \mathbf{P}(t_1)$  are statically applied to the structure and the structural response  $\mathbf{U}_0$  is used as an initial condition for the dynamic structural calculations. The effect of such preliminary static calculation has been found to be very effective to reduce initial transitional effects in the present simulations.

### 3.2. Two-way approach

The two-way simulations take into account the structural motion when calculating the wind flow. As opposed to the one-way approach, these two-way simulations are therefore WSI (or FSI) simulations in the true sense. In addition to the turbulence-induced and part of the wake-induced vibrations, these simulations also capture aeroelastic effects and interactions between the wake-induced excitation and the structural vibration. By comparing the results of the one-way and two-way simulations, it is possible to assess the importance of aeroelastic effects and/or interactions between the wake-induced excitation and the structural motion for the wind-induced vibrations.

In two-way simulations, the structural and flow solver are coupled in every time step as illustrated schematically in figure 6. A partitioned technique is applied which couples the black box flow solver with the black box structural

195 solver. Partitioned means that two independent solvers are used for the flow  
 196 equations and for the structural equations, as opposed to the monolithic ap-  
 197 proach which solves all equations together. Black box means that the source  
 198 code of these solvers is not available and that Jacobians of the outputs with  
 199 respect to the inputs cannot be obtained. Since the resulting aerodynamic  
 200 loads  $\mathbf{P}(t_i)$  and displacements  $\mathbf{U}(t_i)$  typically differ slightly from the ones that  
 201 were applied in the previous iteration within the same time step, consecutive  
 202 flow and structural simulations have to be performed until equilibrium of load-  
 203 ing and displacements is obtained on the interface between structure and wind  
 204 flow.

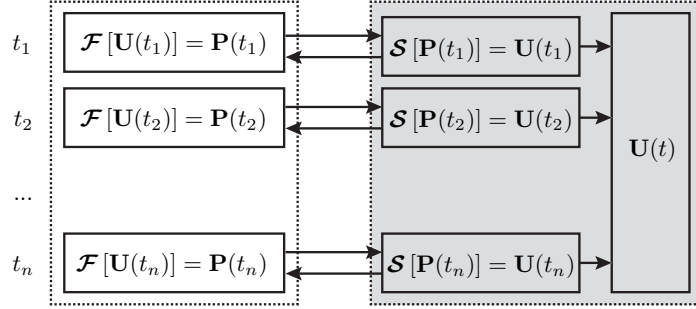


Figure 6: Schematic representation of the interaction between the flow solver  $\mathcal{F}$  (white) and the structural solver  $\mathcal{S}$  (grey) in the two-way partitioned approach. In each time step  $t_i$ , the displacements  $\mathbf{U}(t_i)$  and loading  $\mathbf{P}(t_i)$  on the silo surface are exchanged multiple times between the flow solver and the structural solver. When these coupling iterations have converged, the following time step is started.

205 A multitude of coupling techniques have been proposed to solve FSI prob-  
 206 lems in a partitioned way, as reviewed by Degroote (2013). In each time step,  
 207 these techniques solve the flow equations and the structural equations multiple  
 208 times to find the solution which satisfies the equilibrium conditions. For compu-  
 209 tational efficiency, the number of coupling iterations has to be limited as much as  
 210 possible. Therefore, the interface quasi-Newton technique with an approxima-  
 211 tion for the inverse of the Jacobian of the coupled problem from a least-squares

212 model (IQN-ILS) is applied (Degroote et al., 2009). This root-finding algorithm  
 213 solves the equilibrium equation

$$\mathcal{S}[\mathcal{F}[\mathbf{U}]] - \mathbf{U} = \mathbf{0} \quad (3)$$

214 with Newton-Raphson iterations, using a least-squares approximation for the  
 215 inverse of the Jacobian of this equation. Since wind flow and structural response  
 216 are calculated in black box solvers, this approximation is exclusively based on  
 217 displacements of the fluid-structure interface in previous coupling iterations and  
 218 time steps.

#### 219 **4. Numerical models of wind flow and structure**

220 As discussed in section 3, separate black box solvers can be used for the  
 221 calculation of the wind flow and of the structural response. Details on the  
 222 numerical methods and settings used by these solvers are discussed hereafter.

##### 223 *4.1. Wind flow model*

224 In the 3D CFD model, two different configurations are considered: a single  
 225 silo and a group of 40 silos, all of which are mounted on a prismatic building  
 226 as described in section 2. In the case of the single silo, the building is oriented  
 227 at an angle of incidence of  $\alpha = 45^\circ$  with respect to the incoming wind flow  
 228 while an angle of  $\alpha = 30^\circ$  is considered for the silo group configuration. The  
 229 single silo case is studied in the light of comparing single silo and silo group  
 230 results to assess the importance of the group effect on the ovalling vibrations.  
 231 Furthermore, the single silo is a computationally less expensive problem that  
 232 can be used to study grid sensitivity of the simulations.

233 Several best practice guidelines are available in the literature for the CFD  
 234 simulation of wind flows in urban environments, e.g. the AIJ guidelines (AIJ,  
 235 2006; Tominaga et al., 2008) and the COST guidelines (COST Action C14, 2004;  
 236 COST Action 732, 2007). Not only the size of the computational domain, but  
 237 also the choice of appropriate boundary conditions, mesh and time step size as

well as the choice of convergence criteria for iterative processes are discussed in these documents. These guidelines are followed as closely as possible.

#### *Numerical procedure*

A finite volume method is used in the CFD simulations for the discretization of the governing incompressible Navier-Stokes equations. Large eddy simulation (LES) resolves the large scale turbulent structures in the air flow. However, LES simulations are computationally demanding for high Reynolds number wind flows, especially if near-wall flows are calculated for a good prediction of the aerodynamic forces on the silo walls. By contrast, a Reynolds-averaged Navier Stokes (RANS) model is cheaper to simulate the near-wall flows. As a compromise between these two options, delayed detached eddy simulations (DDES) with  $k - \omega$  SST as RANS model are performed in Ansys Fluent (Fluent 14.5, 2012). This turbulence model has also been used in other studies on similar objects and Reynolds numbers (Squires et al., 2008; Mannini et al., 2011b; Gopalan and Jaiman, 2015). (D)DES models are referred to as hybrid LES/RANS models because the RANS modelling of the boundary layer flow in the near-wall region is combined with the LES approach in the separated regions, where large unsteady turbulence scales are dominant. In the delayed DES approach, a shielding function is used to ensure that RANS is applied in the entire boundary layer since a sole geometrical separation of RANS and LES regions based on mesh size has shown to be insufficient (Spalart et al., 2006). For the shielding function, the blending functions of the SST turbulence model are used (Menter et al., 2003).

For the discretization of the momentum equations a bounded central differencing scheme is used. A second-order interpolation of the pressure, a second-order upwind interpolation of the turbulent kinetic energy  $k$  and the specific dissipation rate  $\omega$  are applied. A bounded second-order implicit, unconditionally stable, time stepping method has been selected. The SIMPLE algorithm (Patankar and Spalding, 1972) is used for the pressure-velocity coupling between the momentum and the continuity equations. In the COST guidelines, it is rec-

ommended that the residuals in the CFD solver should be reduced by at least four orders of magnitude. Therefore, the iterative process in (every coupling iteration within) every time step is truncated when the normalized residuals are all below  $10^{-5}$ .

In the one-way simulations, an Eulerian, fixed grid description is used in the fluid solver. Modifications have to be made to account for a moving fluid grid when a two-way WSI simulation is carried out. Therefore, the flow equations are reformulated in an arbitrary Lagrangian-Eulerian (ALE) description (Donea et al., 1982, 2004). The mesh motion in the wind domain is determined by the motion of the deformable fluid-structure interface. For the simulations with a single silo, a smoothing technique (Batina, 1990; Löhner and Yang, 1996) is used to extend the displacements of the fluid-structure interface into the entire fluid domain. Conversely, for the silo group, Laplace equations are solved for the mesh displacement to handle the more challenging ALE mesh updates in the narrow gaps between adjacent silos. This implies that these diffusion equations for the mesh velocities of the fluid grid have to be solved iteratively in every coupling iteration. Approximately 10 iterations are required for every Laplace mesh update in the present application. This results in good grid quality for all occurring displacements of the silos and causes only a small increase in simulation time compared to the Eulerian formulation.

#### *Computational domain and boundary conditions*

The size of the computational domain is shown in figure 7 and is slightly larger than prescribed by guidelines (AIJ, 2006): e.g. height of the domain  $6H > 5H$ , blockage ratio  $1.7\% < 3\%$ , etc. The dimensions of the domain do not only depend on the zone of interest but also on the applied boundary conditions. The top boundary condition has to sustain the equilibrium ABL profiles. For this purpose, symmetry boundary conditions are applied which provide a good approximation of reality as long as the top boundary is located sufficiently far from the region of interest (COST Action 732, 2007). Symmetry boundary conditions are applied at the lateral boundaries as well while a constant pressure

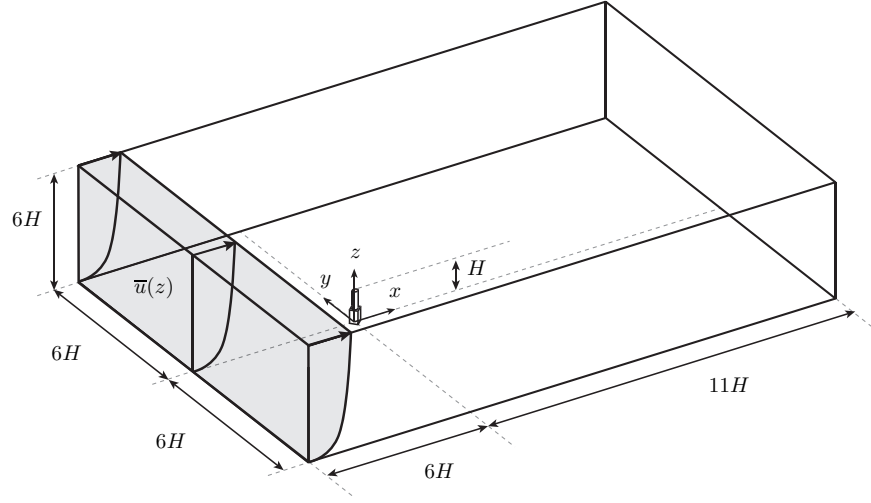


Figure 7: Dimensions of the computational domain and global coordinate system, with origin at the bottom of the domain at the centre of the structure.

is applied at the outflow boundary. The walls of the building and the silo structures are modelled as no-slip walls.

A horizontally homogeneous unidirectional wind flow is applied at the inlet of the domain. Since no exact wind field data are available, approximative wind conditions are set up based on guidelines (AIJ, 2006; BIN, 2010; Tominaga et al., 2008; ANSI, 2010). Given the location of the silo group, the present case is classified in terrain category II of Eurocode 1 (BIN, 2010). The free field wind velocity is set to  $u_{\text{ref}} = 31.8 \text{ m/s}$  at approximately mid-height of the silos, i.e. at  $z_{\text{ref}} \approx 30 \text{ m}$  above ground level. This corresponds roughly to the peak wind velocities that were monitored in 2002 at the meteorological station in Deurne a few kilometers from the site. The highly turbulent wind flow regime around the group of silos is hence post-critical (Zdravkovich, 1997) at Reynolds number  $\text{Re} = u_{\text{ref}} D / \nu = 1.24 \times 10^7$  with  $\nu = 1.41 \times 10^{-5} \text{ m}^2/\text{s}$ .

For the variation of the mean wind velocity with height  $\bar{u}(z)$  in the atmospheric boundary layer (ABL, cfr. shaded area in figure 7), a power law profile as prescribed in the AIJ guidelines is used (AIJ, 2006; Tominaga et al., 2008).

314 The mean wind velocity  $u_{\text{ref}}$  at a reference height of  $z_{\text{ref}} = 30 \text{ m}$  is used to fit  
 315 the power law:

$$\overline{u}(z) = u_{\text{ref}} \left( \frac{z}{z_{\text{ref}}} \right)^{\beta}, \quad (4)$$

316 with  $\beta$  a constant depending on the terrain roughness which equals 0.14 for the  
 317 considered terrain category (II, open country).

318 Note that an incompatibility of inflow data and wall boundary conditions  
 319 at ground level may have an important impact on the preservation of the ho-  
 320 mogeneity of the ABL flow (Blocken and Carmeliet, 2006; COST Action C14,  
 321 2004; COST Action 732, 2007; Quinn et al., 2001; Zhang, 1994). This effect is  
 322 limited to the lowest region of the ABL however (Castro, 2003). In the German  
 323 guidelines (VDI-Richtlinie 3783, 2003), it is therefore recommended to place at  
 324 least two nodes between the ground and the zone of interest. Considering the  
 325 fact that the silo structures are mounted on a prismatic building, this effect is  
 326 of no further consequence in the present simulations.

327 To simulate a 3D turbulent wind flow, realistic random wind field data have  
 328 to be modelled at the inlet of the domain as well. In the present simulations,  
 329 turbulence is synthetically generated with a spectral synthesizer method pro-  
 330 posed by Smirnov et al. (2001), as implemented in Ansys Fluent (Fluent 14.5,  
 331 2012). A power law profile for the turbulence intensity  $I_u(z)$ , based on the AIJ  
 332 guidelines (Tominaga et al., 2008) is used as input for this spectral synthesizer  
 333 method:

$$I_u(z) = 0.1 \left( \frac{z}{z_G} \right)^{-\beta-0.05}, \quad (5)$$

334 with  $z_G = 350 \text{ m}$  the gradient height of the ABL in open country (terrain  
 335 category II) (Tominaga et al., 2008).

### 336 *Discretization sensitivity analysis*

337 In the case of DDES simulations, it is advisable to get an indication of the  
 338 sensitivity of the results to discretization cut-offs. In tables 1 and 2, the size of  
 339 the computational domain is listed for several levels of grid refinement in the  
 340 simulation of the flow around the single silo and around the entire silo group,



341 respectively. Most of the grid is block-structured, except for the part at the  
 342 top of the silos. The maximal  $y^+$  values along the circumference are around  
 343 100, so wall functions are applied. At least 60 cells are present in the gaps  
 344 between the silos of the group. It is obvious that an increase in number of grid  
 345 cells results in a significant increase of computational efforts. It takes about 40,  
 346 72 and 111 hours of computing time for the coarse, medium and fine meshes,  
 347 respectively, to calculate 40 s of wind flow in the 3D simulation for a single silo.  
 348 These calculations have been performed on 64 parallel cores, i.e. half of the cores  
 349 of four quad-socket octa-core AMD Magny-Cours machines (Opteron 6136) at  
 350 2.4 GHz with almost full use of the working memory of all machines (64 GB  
 351 RAM per machine). For the simulation of the wind flow around the entire silo  
 352 group on the coarse and fine grids, respectively 280 and 450 hours are required  
 353 to simulate the same time frame of 40 s on the same 64 parallel cores.

Mesh	$N_{\text{circ}}$	$N_{\text{dom}}$
coarse	64	1 573 872
medium	80	4 552 486
fine	112	8 233 532

Table 1: Overview of the number of cells  $N_{\text{dom}}$  in the computational domain for the single silo  
 simulations. Three stages of grid refinement are considered, based on the number of divisions  
 $N_{\text{circ}}$  on the circumference of the silo surface.

Mesh	$N_{\text{circ}}$	$N_{\text{dom}}$
coarse	64	8 276 472
fine	80	17 066 268

Table 2: Overview of the number of cells  $N_{\text{dom}}$  in the computational domain for the silo  
 group simulations. Two stages of grid refinement are considered, based on the number of  
 divisions  $N_{\text{circ}}$  on the circumference of the silo surface.

354 In view of the objectives of the present investigation, this inevitably implies

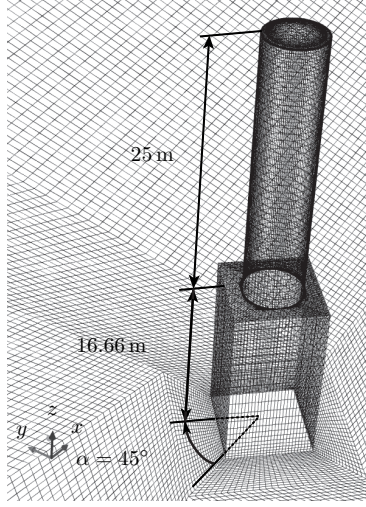


Figure 8: Detail of the coarse mesh of a single silo structure with the wind at an angle of incidence  $\alpha = 45^\circ$ .

that compromises between accuracy and required computational effort have to be made, especially for the two-way simulations where multiple coupling iterations have to be performed per time step. In this respect, it is important to assess discretization errors and take them into account for the interpretation of simulation results rather than pursuing extreme grid refinements to obtain a completely grid independent solution. Furthermore, it is noted that a simulated time frame of only 40 s is fairly short in wind engineering where typically response quantities are averaged over periods of 10 minutes or longer. Again, the length of these time intervals is limited by the high computational costs.

First, the single silo configuration is considered. A detail of the computational mesh (coarse grid) for the single silo is shown in figure 8, with incoming flow along the  $x$ -axis. The results for the mean and RMS values of the drag coefficients in  $x$ - and  $y$ -directions are summarized in table 3. A variability in the drag coefficients for the different grid sizes is observed. These discrepancies are due to differences in the predicted location of the separation point on the silo surface and, hence, the largest differences are observed for the mean drag

coefficient  $\overline{C}_{dx}$  in the  $x$ -direction. It should be underlined that the accurate prediction of separation points on smooth, curved surfaces is a major challenge for every turbulence modelling technique in CFD.

Mesh	$\overline{C}_{dx}$	$C_{dx}^{\text{RMS}}$	$\overline{C}_{dy}$	$C_{dy}^{\text{RMS}}$
Coarse	0.426	0.021	0.025	0.061
Medium	0.359	0.018	0.018	0.052
Fine	0.302	0.017	-0.005	0.027

Table 3: Mean value and root mean square value of the drag coefficients  $C_{dx}$  and  $C_{dy}$  for the single silo with wind flow at an angle of incidence  $\alpha = 45^\circ$  and for different mesh refinements.

For a selection of representative silos in the group, the drag coefficients in  $x$ - and  $y$ -directions are listed in table 4. Figure 9 furthermore shows the time history of  $C_{dx}$  and  $C_{dy}$  for the corner silos of the group. From table 4, it is observed that the cylinders at the front of the silo group (e.g. silos 1, 17, 25 and 33) are subject to larger drag forces than silos at the centre of the group (e.g. silos 30 and 38). Silos 1, 9, 17, 25 and 33 have the largest area exposed directly to the incoming flow (figure 2), resulting in the highest drag coefficients  $C_{dx}$ . Overall, it is concluded that the drag coefficients predicted on both meshes are in reasonably good agreement for the majority of the silos, including for silo 1 where the ovalling vibrations have been observed. Although the separation of the flow is much more geometrically determined for the entire silo group, difficulties with the prediction of separation are still observed for the transverse corner silos (silos 8 and 33 in figure 2) and for some of the upwind silos in the first row of the group. This difference between the results on both meshes at the transverse corner silos is mainly due to the prediction of the separation points on smooth, curved surfaces as in the single silo case.

Finally, also the influence of the time step on the accuracy of the simulations has been assessed by performing different simulations with a systematic reduction of the time step. This time step refinement has only been carried out for the single silo configuration. From the results summarized in table 5,

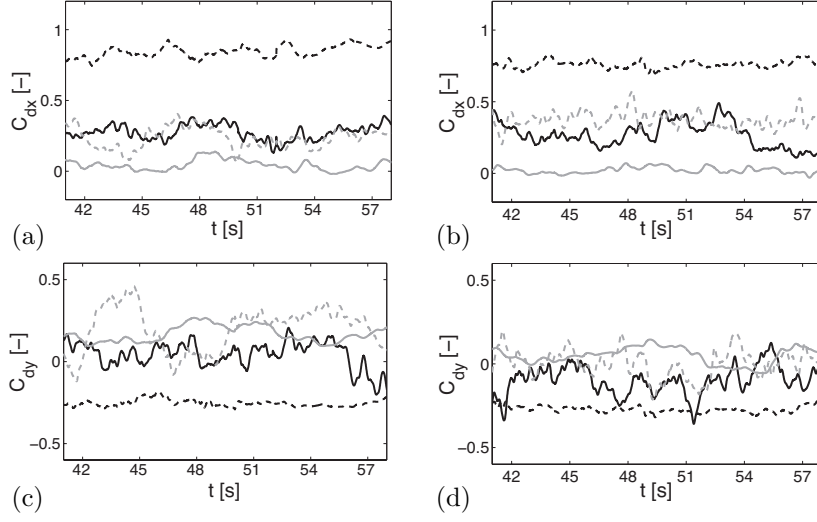


Figure 9: Time history of the drag coefficients (top)  $C_{dx}$  and (bottom)  $C_{dy}$  for the corner silos of the silo group (silo 1 - dashed black line; silo 8 - solid black line; silo 33 - dashed grey line; silo 40 - solid grey line) with wind flow at an angle of incidence  $\alpha = 30^\circ$  and for (left) the coarse mesh and (right) the fine mesh.

it is concluded that a time step  $\Delta t = 0.005$  s is sufficiently small. Although no explicit verification was made for the entire silo group, it can be reasoned that all physical flow phenomena for the silo group have larger length scales and hence longer periods which can be accurately captured by this time step. The COST guidelines (COST Action C14, 2004; COST Action 732, 2007) instruct that the choice of the time step is determined by the relevant frequencies in the flow: the highest frequency to be considered needs to be resolved with at least 10 to 20 time steps. When a time step size of 0.005 s is used, this means that frequencies of 10 Hz are still resolved. In the following, it will be shown that this is sufficiently accurate given the low natural frequencies of the silo structures and the low frequency content of the wind flow.

#### 4.2. Structural model

The software package Abaqus (Abaqus 6.10, 2010) has been used to construct a finite element (FE) model of the silo structure which allows to calculate the

Silo	Mesh	$\overline{C}_{dx}$	$C_{dx}^{RMS}$	$\overline{C}_{dy}$	$C_{dy}^{RMS}$
1	Coarse	0.841	0.045	-0.253	0.022
	Fine	0.781	0.043	-0.268	0.022
5	Coarse	0.221	0.048	0.199	0.045
	Fine	0.233	0.057	0.192	0.043
8	Coarse	0.409	0.121	-0.094	0.154
	Fine	0.361	0.128	-0.269	0.195
17	Coarse	0.757	0.053	-0.381	0.031
	Fine	0.798	0.057	-0.411	0.045
25	Coarse	0.704	0.055	-0.269	0.043
	Fine	0.748	0.061	-0.281	0.050
30	Coarse	0.036	0.030	0.037	0.017
	Fine	0.038	0.031	0.043	0.024
33	Coarse	0.254	0.114	0.175	0.166
	Fine	0.403	0.072	0.059	0.096
38	Coarse	0.069	0.035	0.066	0.028
	Fine	0.043	0.034	0.064	0.033
40	Coarse	0.050	0.046	0.152	0.061
	Fine	0.042	0.048	0.104	0.066

Table 4: Mean value and root mean square value of the drag coefficients  $C_{dx}$  and  $C_{dy}$  for a selection of silos of the 8 by 5 silo group with wind flow at an angle of incidence  $\alpha = 30^\circ$  and for the coarse and the fine mesh. The silos on the corners of the group are highlighted in the table.

structural response of the silos to applied aerodynamic pressures and shear stresses. This model is also used to determine the ovalling eigenmodes and corresponding natural frequencies of the silos.

Each silo in the group is made of 10 aluminium cylindrical sheets with a height of 2.5 m each (figure 10). At the top and bottom of the cylinder, a cone is welded to the cylinder at an angle of  $15^\circ$  and  $60^\circ$  with the horizontal plane, re-

$\Delta t$	$C_{dx}^{\text{RMS}}$	$\Delta C_{dy}$
0.01 s	0.022	0.104
0.005 s	0.016	0.085
0.0025 s	0.016	0.082
0.000125 s	0.016	0.082

Table 5: Root mean square value of drag coefficient  $C_{dx}$  and amplitude  $\Delta C_{dy} = C_{dy}^{\text{max}} - C_{dy}^{\text{min}}$  for the 3D single silo (fine mesh) and for different time steps.

spectively. Shell elements with linear FE interpolation functions are used for the entire structure. For the dynamic simulations, the unconditionally stable and second-order accurate Hilbert-Hughes-Taylor (HHT) method as implemented in the Abaqus FE solver is used.

The following material properties are used for aluminium: density  $\rho = 2700 \text{ kg/m}^3$ , Young's modulus  $E = 67.6 \text{ GPa}$  and Poisson's ratio  $\nu = 0.35$ . Rayleigh damping is assumed and based on a constant damping ratio  $\xi = 0.75\%$  for the two lowest eigenmodes. This is a realistic approximation since modal damping ratios  $\xi$  for this structure, determined during in situ measurements by Dooms et al. (2006), were found to vary between 0.07% and 1.32%. These low modal damping values are typical for a welded aluminium structure. The silo is bolted to an octagonal steel framework at 4 points on the circumference of the cylindrical shell (figure 10).

To accommodate an easy transfer of the aerodynamic loads on the silo walls from flow solver to structural solver, the mesh of the FE model is chosen conforming to the mesh on the silo walls in the CFD simulations. As a result, three different structural models are used, i.e. one for each of the three CFD grids for a single silo (cfr. table 1).

#### *Natural frequencies and ovaling modes*

The mass normalized eigenmodes  $\Phi$  and corresponding eigenfrequencies  $f_{\text{eig}}$  are determined by solving the generalized eigenvalue problem of the structure.

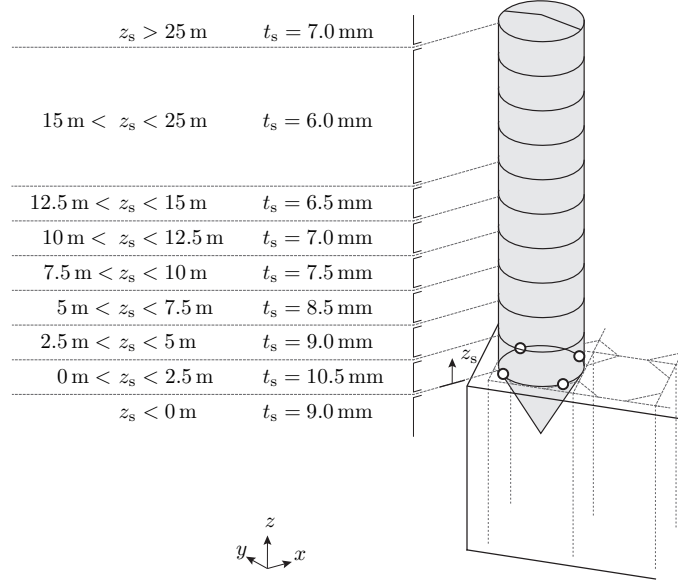


Figure 10: Geometrical representation of the silo structure with indication of the variation of the shell thickness  $t_s$  of the aluminium plates along the height  $z_s$  of the silo structure.

435 The eigenmodes corresponding to the lowest eigenfrequencies are summarized  
 436 in table 6 and a selection of these modes is shown in figure 11.

$\Phi_j$	$(m, n)$	$f_{\text{eig}j} [\text{Hz}]$	$\Phi_j$	$(m, n)$	$f_{\text{eig}j} [\text{Hz}]$	$\Phi_j$	$(m, n)$	$f_{\text{eig}j} [\text{Hz}]$
$\Phi_1$	(1, 3)	3.96	$\Phi_7$	(1, 5)	5.70	$\Phi_{13}$	(2, 5)	8.19
$\Phi_2$	(1, 3)	3.97	$\Phi_8$	(1, 5)	5.71	$\Phi_{14}$	(2, 6)*	8.62
$\Phi_3$	(1, 4)	3.99	$\Phi_9$	(1, 6)	7.72	$\Phi_{15}$	(2, 4)	8.85
$\Phi_4$	(1, 4)	4.11	$\Phi_{10}$	(1, 6)	7.72	$\Phi_{16}$	(2, 4)	9.10
$\Phi_5$	(1, 5)	5.34	$\Phi_{11}$	(1, 2)	7.83	$\Phi_{17}$	(2, 6)	9.62
$\Phi_6$	(1, 5)	5.35	$\Phi_{12}$	(2, 5)	8.18	$\Phi_{18}$	(2, 6)	9.72

Table 6: Natural frequencies  $f_{\text{eig}}$  of the lowest ovalling eigenmodes of the silo structure. The mode shapes indicated with an asterisk are a combination of two ‘pure’ ovalling shapes.

437 The mode shapes are of the ovalling type and are referred to by a couple  
 438  $(m, n)$ , where  $m$  denotes the number of half wavelengths in the axial direction

and  $n$  is the number of circumferential wavelengths. For an axisymmetric structure most of the ovalling eigenmodes come in pairs: e.g.  $\Phi_1$  and  $\Phi_2$  are both classified as mode shapes (1,3) but are mutually orthogonal. The mode shapes indicated with an asterisk, e.g.  $\Phi_{14} = (2,6)^*$  are referred to as ‘hybrid’ mode shapes. Such mode shapes are characterized by two combined ‘pure’ ovalling shapes at once, e.g. shapes (1,2) and (2,6) in the case of  $\Phi_{14} = (2,6)^*$  (figure 11d).

From the silo group vibrations observed during the storm in 2002, it is believed that mainly ovalling modes (1,3) and (1,4) were excited by the wind field, corresponding to the lowest eigenfrequencies of the silo structure. Measurements during normal wind loading have also shown that eigenmodes with 3 or 4 circumferential wavelengths have the highest contribution to the response of the silos (Dooms et al., 2006).

For the physical interpretation of the structural response, a more quantitative analysis is proposed in the following that allows to assess the modal contributions in the response. The deformation energy  $E_d(t)$  is easily determined from the calculated structural response  $\mathbf{U}(t)$  of the entire structure:

$$E_d(t) = \frac{1}{2} \mathbf{U}^T(t) \mathbf{K} \mathbf{U}(t), \quad (6)$$

where  $\mathbf{K}$  represents the stiffness matrix of the structural FE model. The contributions of the different eigenmodes in the response are subsequently distinguished by applying modal decomposition. By inserting  $\mathbf{U}(t) = \Phi \boldsymbol{\alpha}(t)$ , with  $\boldsymbol{\alpha}(t)$  the modal coordinates, in equation (6), the deformation energy is decomposed into modal contributions as follows:

$$E_d(t) = \frac{1}{2} \boldsymbol{\alpha}^T(t) \Phi^T \mathbf{K} \Phi \boldsymbol{\alpha}(t) = \frac{1}{2} \sum_{j=1}^{n_{\text{DOF}}} \omega_j^2 \alpha_j^2(t) = \sum_{j=1}^{n_{\text{DOF}}} E_{dj}(t) \quad (7)$$

where  $n_{\text{DOF}}$  is the number of degrees of freedom of the FE model. Based on this expression, the contribution  $E_{dj}(t)$  of an individual mode  $j$  to the deformation energy  $E_d(t)$  can be determined from the modal coordinates  $\boldsymbol{\alpha}(t)$ .



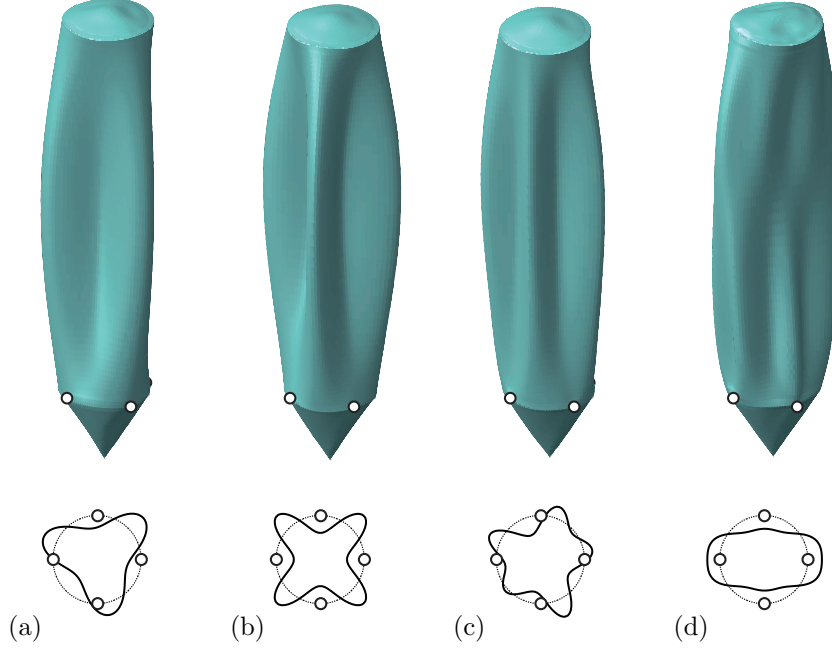


Figure 11: (top) 3D isotropic view and (bottom) horizontal section at mid-height for a selection of ovaling eigenmodes of a silo: (a) mode  $\Phi_1 = (1, 3)$  at 3.96 Hz, (b) mode  $\Phi_4 = (1, 4)$  at 4.11 Hz, (c) mode  $\Phi_5 = (1, 5)$  at 5.34 Hz and (d) mode  $\Phi_{14} = (2, 6)^*$  at 8.62 Hz.

## 464 5. Wind-structure interaction simulation results

465 In this section, the wind-induced vibrations of the silos are analyzed using  
 466 wind-structure interaction simulations. As described in section 3 both one-way  
 467 and two-way simulations are performed for the single silo and the silo group  
 468 configuration. Given the agreement between the results on the different grids  
 469 in the previous section, the second finest grid is presented here. Simulations  
 470 performed on other grids led to the same conclusions and are hence omitted for  
 471 brevity. A time frame with the last 25 s is considered in all coupled simulations  
 472 unless indicated differently.

### 473 5.1. *Single silo simulations*

#### 474 *One-way simulations*

475 First, the flow pattern around the single silo as shown in figure 12 is consid-  
476 ered. Streamlines are shown in a vertical and several horizontal planes across  
477 the height of the structure. The flow has a highly three-dimensional character  
478 and different flow patterns can be observed at different heights. Upstream of  
479 the structure, near the ground, a horseshoe vortex is clearly formed in the flow  
480 (figures 12top and a). The flow around the prismatic building is attached to  
481 the surface on the windward side of the prism and separation occurs at the  
482 corners of the building. At the connection between the prismatic building and  
483 the cylindrical silo structure, the flow is deflected upwards and detaches from  
484 the sharp edges of the building. These accelerated upward flows are deflected  
485 sideways due to the vicinity of the cylindrical silo (figure 12b). Along the height  
486 of the silo structure, the flow is separated at the lee side of the cylindrical sur-  
487 face, generating a highly turbulent and narrow wake region, typical for high  
488 Reynolds number cross-flows around cylinders. At the free end of the silo struc-  
489 ture (figure 12d), the separated trailing vortex is mainly dragged downstream  
490 and only slightly deflected downwards. The existing downwash effect has al-  
491 ready disappeared at mid-height of the silo (figure 12c). As indicated by Park  
492 and Lee (2004), the conical shape of the silo top might also explain the reduced  
493 width of the wake formed near the free end of the cylinder. Downwash effects  
494 are therefore thought to be limited at the present Reynolds number and little if  
495 any interaction with the flow at midspan of the silo is observed. It could also be  
496 argued that downwash effects can only have a very limited effect on the already  
497 narrow and short wake region over the entire length of the silo structure.

498 In the one-way simulation of the single silo, a predominantly static structural  
499 response with maximal displacements of approximately 0.04 m is found and  
500 significantly smaller ovaling vibrations in the order of 0.01 m are observed.  
501 These deformations are converted into nodal contributions to the deformation  
502 energy  $E_{dj}(t)$  which are shown as a function of time in figure 13 for the 20

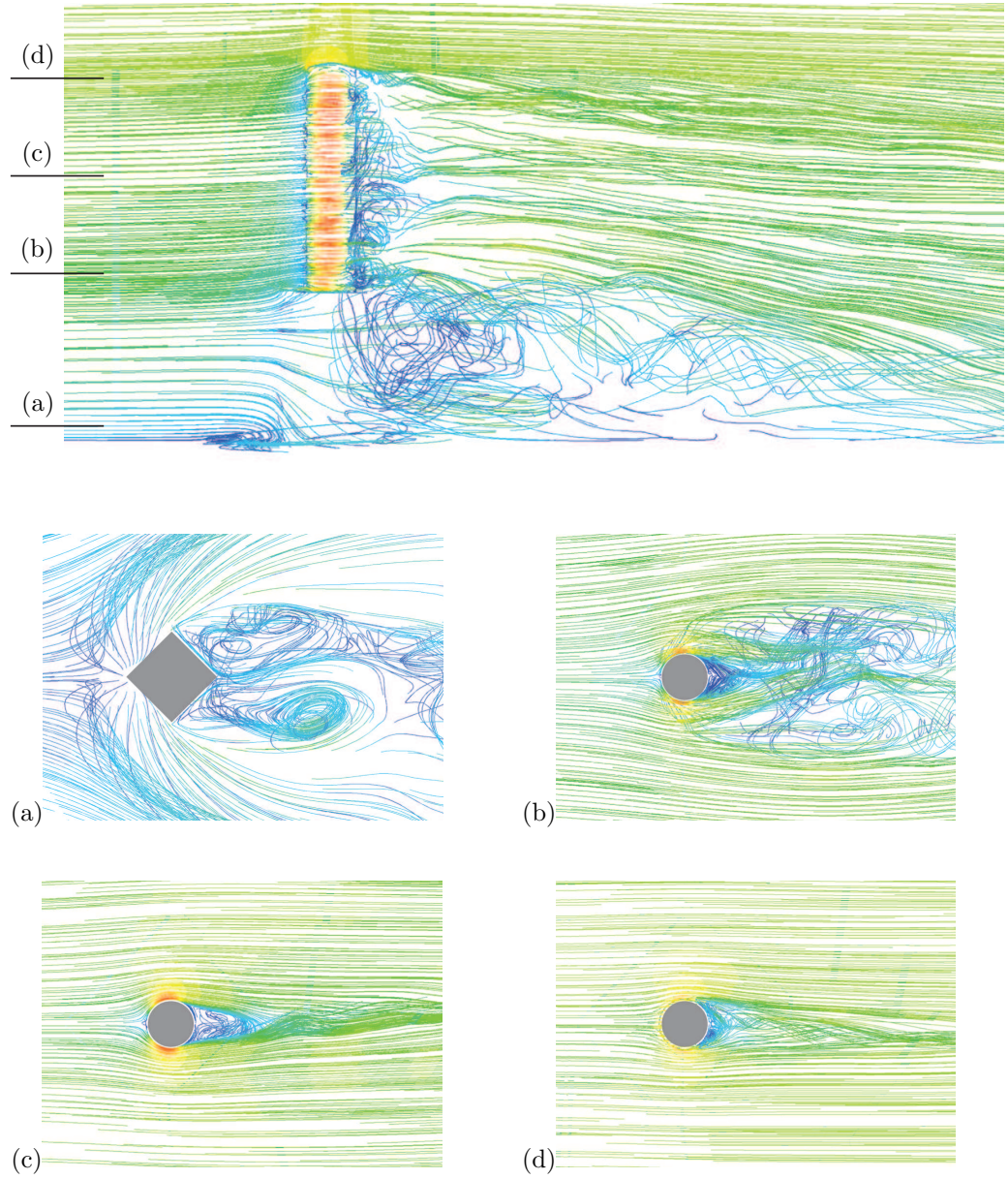


Figure 12: Streamlines of the wind flow around a single silo structure after 40 s, colored according to the velocity magnitude, released from (top) a vertical plane  $y = 0$  m and horizontal planes at (a)  $z = 0.5$  m, (b)  $z = 17.16$  m, (c)  $z = 29.16$  m, and (d)  $z = 41.16$  m.

503 structural eigenmodes with the lowest eigenfrequencies. Additional calculations  
 504 accounting for geometrical nonlinearities did not reveal a significant effect of  
 505 nonlinearities on the structural response.

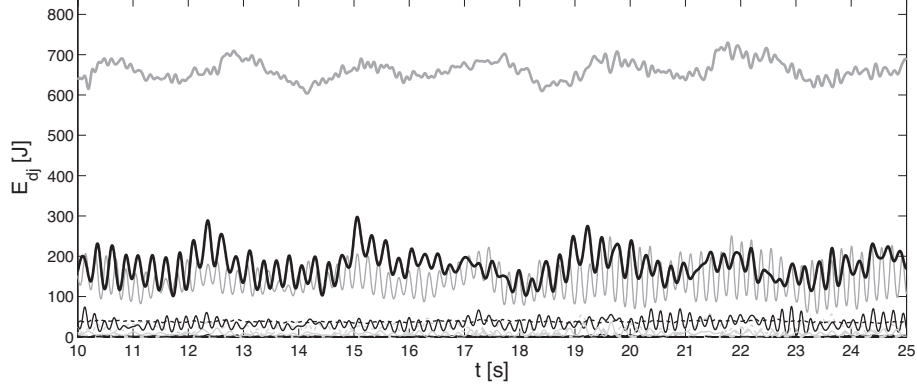


Figure 13: Modal deformation energy  $E_{dj}(t)$  of the first 20 mode shapes, computed from the structural response in the one-way simulation of a single silo:  $\Phi_2 = (1, 3)$  (bold black line),  $\Phi_4 = (1, 4)$  (thin black line),  $\Phi_6 = (1, 5)$  (thin grey line),  $\Phi_{14} = (2, 6)^*$  (bold grey line),  $\Phi_{18} = (2, 6)$  (dashed black line), and the remaining mode shapes  $\Phi_j$  (dashed grey lines, with small energy content).

506 Figure 13 shows that only a limited number of modes contribute significantly  
 507 to the structural response of the silo. Typically, three time scales, named mean,  
 508 resonant and quasi-static, are distinguished in the modal deformation energy  
 509  $E_{dj}(t)$  (Holmes, 2015).

- 510 • The mean, time averaged modal deformation energy is related to the static  
 511 excitation of a structural mode. Mode  $\Phi_{18} = (2, 6)$ , for example, is almost  
 512 exclusively excited statically (dashed black line in figure 13).
- 513 • While a whole range of irregular oscillations seems to be present in the  
 514 fluctuating parts of the response, regular oscillations of eigenmodes  $\Phi_2 =$   
 515  $(1, 3)$  (bold black line),  $\Phi_4 = (1, 4)$  (thin black line) and  $\Phi_6 = (1, 5)$  (thin  
 516 grey line) at their respective eigenfrequencies can be clearly observed.  
 517 This resonant response is mainly found for the eigenmodes with the lowest

518 natural frequencies.

- 519 • All remaining oscillations are categorized as the background response of  
520 the structure. The fourteenth mode  $\Phi_{14} = (2, 6)^*$ , for example, is almost  
521 exclusively excited by low frequency fluctuations (bold grey line in fig-  
522 ure 13). Because this is a one-way simulation, it is sure that these low  
523 frequency fluctuations originate from the wind, but it is difficult to dis-  
524 cern between incoming flow and the silo wake. Such oscillations typically  
525 give rise to quasi-static swaying deformations of the silo, free of resonant  
526 effects.

527 In the following, time averaged results are considered to facilitate comparison  
528 of results for one-way and two-way coupled simulations. Figures 14a and b show  
529 the time averaged  $\overline{E}_{dj}$  and RMS values  $E_{dj}^{\text{RMS}}$  of the modal deformation energy  
530 for the one-way simulation of the single silo for the 50 eigenmodes with the lowest  
531 eigenfrequencies. The mean modal deformation energy  $\overline{E}_{dj}$  (figure 14a) gives  
532 an indication of the static excitation of the eigenmodes while the RMS values  
533  $E_{dj}^{\text{RMS}}$  (figure 14b) contain information on resonant and quasi-static motion.

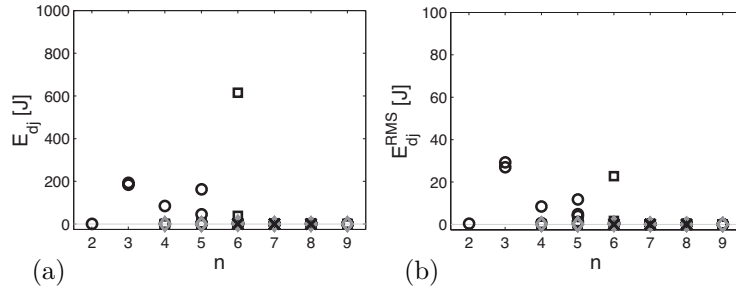


Figure 14: (left) Mean value and (right) RMS value of the modal contributions to the deformation energy  $E_{dj}(t)$  for the single silo in the one-way simulation. The deformation energy for the lowest 50 eigenmodes is plotted as a function of  $n$  while separate mode shapes  $(1, n)$  are depicted as a circle (o),  $(2, n)$  as a square (□),  $(3, n)$  as a diamond (◇), and  $(4, n)$  as a cross (×).

534 The mean modal deformation energies in figure 14a are significantly larger

than the RMS values in figure 14b (shown on a different vertical scale). From these figures, it is observed that modes  $\Phi_2 = (1, 3)$ ,  $\Phi_6 = (1, 5)$  and  $\Phi_{14} = (2, 6)^*$  are predominantly excited statically. This is of course related to the aerodynamic forces applied on the structure: high pressures at stagnation, low pressures in the attached boundary layer flows on the sides and a more or less constant pressure after separation. A snapshot of the resulting displacements is shown in figure 15a.

By comparing figures 14a and 14b for the mean and RMS value of the modal deformation energy, respectively, it is clear that structural vibrations are much smaller than the static displacements. According to figure 14b, the mode shapes that are excited dynamically are the eigenmodes with the lowest circumferential wavenumber  $n$  and only a half wavelength across the height ( $m = 1$ ). It is noted that the RMS values characterize the magnitude of the dynamic response, regardless of the nature of the latter. For example, the RMS value of mode  $\Phi_{14} = (2, 6)^*$  is due to quasi-static vibration while for modes  $\Phi_2 = (1, 3)$  and  $\Phi_6 = (1, 5)$  RMS values similar in magnitude correspond to resonant vibrations. This can be observed in figure 13 where the amplitudes of the oscillations at the corresponding eigenfrequencies are smaller for the former than for the latter two.

#### *Two-way simulations*

Two-way simulations are performed for the single silo to assess the change in structural response due to aeroelastic effects and/or interaction between wake-induced excitation and the vibration. In general, 5 IQN-ILS coupling iterations have to be performed in each time step to ensure equilibrium on the WSI interface. The computational effort required for the two-way simulations is therefore approximately 5 times larger than for the one-way simulations.

The time histories of the modal deformation energy in the one-way (figure 13) and the two-way simulation (figure 16) are compared. Furthermore, the averaged and RMS values of the response in the one-way and two-way simulation are compared (figures 14 and 17). It is observed that the static response and

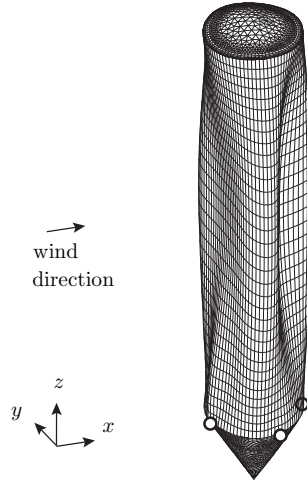


Figure 15: Displacement of the single silo at  $t = 15$  s in the one-way simulation. The structural displacements are mainly static and are amplified with a scale factor of 40.

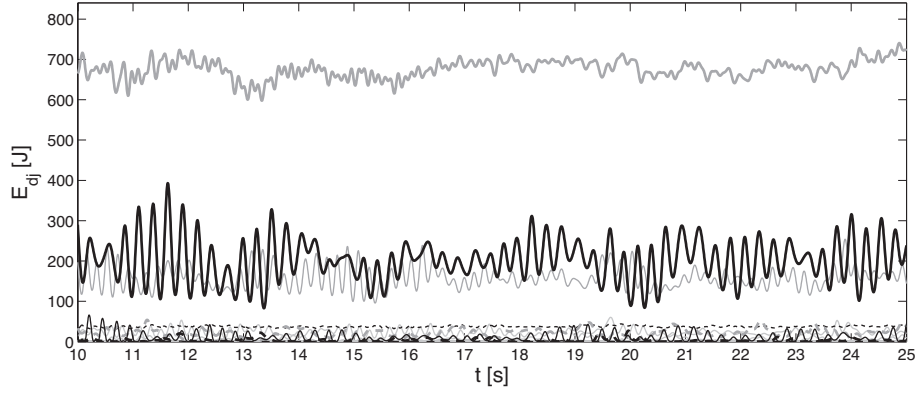


Figure 16: Modal deformation energy  $E_{dj}(t)$  of the first 20 mode shapes, computed from the structural response in the two-way coupled simulation of a single silo:  $\Phi_2 = (1, 3)$  (bold black line),  $\Phi_4 = (1, 4)$  (thin black line),  $\Phi_6 = (1, 5)$  (thin grey line),  $\Phi_{14} = (2, 6)^*$  (bold grey line),  $\Phi_{18} = (2, 6)$  (dashed black line), and the remaining mode shapes  $\Phi_j$  (dashed grey lines, with small energy content).

565 the RMS value of the deformation energy are similar in both simulations. This  
566 is reflected in the similar static peak displacements of approximately 0.04 m in

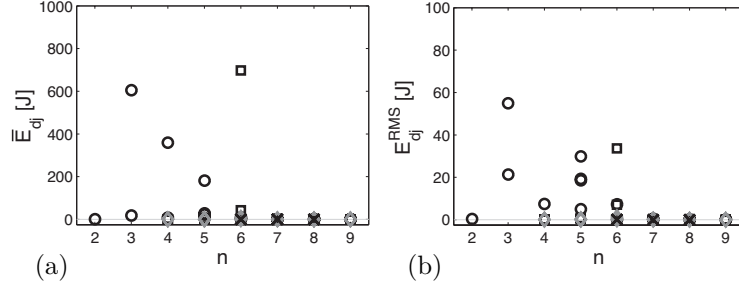


Figure 17: (left) Mean value and (right) RMS value of the modal contributions to the deformation energy  $E_{dj}(t)$  for the single silo in the two-way simulation. The deformation energy for the lowest 50 eigenmodes is plotted as a function of  $n$  while separate mode shapes  $(1, n)$  are depicted as a circle ( $\circ$ ),  $(2, n)$  as a square ( $\square$ ),  $(3, n)$  as a diamond ( $\diamond$ ), and  $(4, n)$  as a cross ( $\times$ ).

both simulations. For eigenmode  $\Phi_2 = (1, 3)$ , the static and RMS value of the dynamic response are distributed differently over the two orthogonal mode shapes in the one-way and two-way simulations but with a similar total value.

In both the one-way and the two-way simulations, the RMS values of the deformation energy are one order of magnitude smaller than the corresponding static values. It is concluded from these observations that qualitatively similar results are found in the one-way and two-way simulation of a single silo, with the same modes contributing to the quasi-static and resonant response of the structure.

## 5.2. Silo group simulations

In order to study the effect of the group arrangement on the wind-induced vibrations, both one-way and two-way simulations have been performed for silos at different locations in the silo group. In the one-way simulations, all corner silos are flexible while in the two-way simulation, only the silo at the windward corner is flexible as this is where the largest vibration amplitudes were observed. The same methodology is applied as in the single silo case.



583 *One-way simulations*

584 Velocity streamline plots of the flow pattern around the silo group are given  
585 in figure 18. It is obvious that the flow pattern around the entire silo group has  
586 a much more complex and turbulent nature than that around the single silo.  
587 Nevertheless, some trends that were previously discussed are also observed for  
588 the group configuration. The formation of a horseshoe vortex is for instance  
589 clearly observed (figures 18top and a). The horseshoe vortex is observed at the  
590 same location as in Shah and Ferziger (1997) for the wind flow around a cube  
591 building with one face perpendicular to the incident wind. In this study, it  
592 was found that the primary separation occurs at a saddle point located about  
593 one obstacle height ahead of the obstacle. The horseshoe vortex is also much  
594 higher than for the single silo configuration, deflecting the approaching wind  
595 flow upwards from a larger upstream distance. Separation occurs at the sharp  
596 edges at the transverse corners of the building, generating a large wake region  
597 with irregular recirculation zones. In the region where the silos are located,  
598 a generally similar pattern can be observed but comparing figures 18b and c  
599 reveals some important differences. Although the flow is only partly attached  
600 to the silo structures at the windward side of the silos with a small part of  
601 the flow entering the silo array, the global picture remains similar as for the  
602 bluff prismatic building where all flow is deflected sideways. At the transverse  
603 corners of the silo group (silos 8 and 33), however, flow separation is only partly  
604 geometrically triggered and separation occurs on the silo surfaces. At the top  
605 of the silo group, the flow is deflected upwards and sideways. It only slightly  
606 accelerates but separates from the edges at the top (figures 18top and d). No  
607 specific point of reattachment on the top can be observed as for simple block-like  
608 geometries in wind flow. This is mainly attributed to the open structure of the  
609 silo array with wind flows emerging from the interstitial spaces and interaction  
610 with the conical shape of the 40 silo tops. Similarly as for the single silo case,  
611 the size of the wake is reduced near the free end of the group but the effect is  
612 more limited. The wake behind the silo group is reduced less, creating a larger

613 flow resistance as compared to the single silo case.

614 The influence of the location of a silo in the group on the structural response  
615 is investigated by considering the four corner silos: 1, 8, 33 and 40 (figure  
616 2). Additional simulations showed that the response at these locations was  
617 representative for all windward and leeward silos in the group.

618 First, the structural response of silo 1 at the windward corner of the silo  
619 group is calculated. The mean and RMS values of the modal deformation energy  
620 are shown in figures 19a and b, respectively. It can be clearly observed that the  
621 static deformation of this silo is dominated by eigenmodes (1, 3) and (1, 4) (figure  
622 19a). Other eigenmodes with low circumferential wavenumbers ( $n$ ) are also  
623 excited, but less pronounced. Roughly the same eigenmodes are also excited  
624 dynamically as demonstrated by the RMS values of the modal deformation  
625 energy (figure 19b). The RMS values for silo 1 are slightly larger than for the  
626 single isolated silo. Note the different vertical scale of the RMS graphs compared  
627 to the simulations for a single silo (figure 14).

628 Predominant static and dynamic excitation of modes (1, 3) and (1, 4) is also  
629 seen in the response snapshots in figure 20. It is observed that the silo wall is de-  
630 flected inwards due to the large positive pressures at the windward side. When  
631 moving downstream along the silo wall, the attached boundary layer causes  
632 suction and the shell is deformed outwards. The largest structural displace-  
633 ments, however, are found in the small gaps between two adjacent silos. Due to  
634 the larger wind velocities in these narrow passages, negative surface pressures  
635 develop and the silos deform accordingly. The magnitude of the structural dis-  
636 placements is as high as 0.07 m in these interstitial spaces. These displacement  
637 values are large in comparison with the total distance of only 0.3 m between two  
638 neighbouring silos. The vibration amplitudes, however, are significantly smaller  
639 (approx. 0.01 to 0.02 m).

640 For the silos on the transverse corners of the silo group, i.e. silos 8 and 33,  
641 the results of the one-way simulations are shown in figures 19c to f. For silos 8  
642 and 33, the RMS value of the modal contributions is significantly larger than for  
643 a single silo or silo 1 in the group. This is in agreement with the time history of

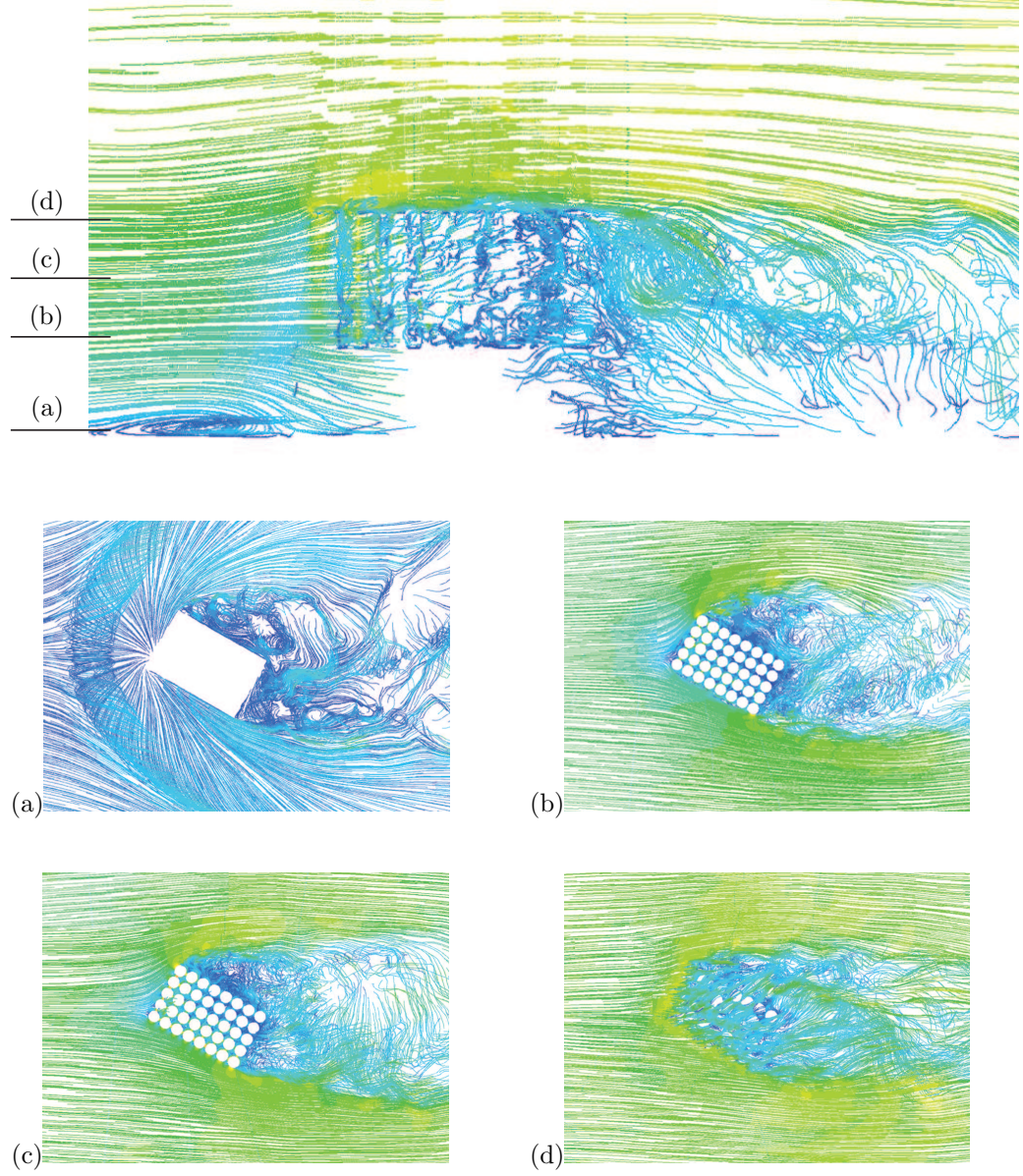


Figure 18: Streamlines of the wind flow around the silo group after 40 s, colored according to the velocity magnitude, released from (top) a vertical plane  $y = 0$  m and horizontal planes at (a)  $z = 0.5$  m, (b)  $z = 17.16$  m, (c)  $z = 29.16$  m, and (d)  $z = 41.16$  m.

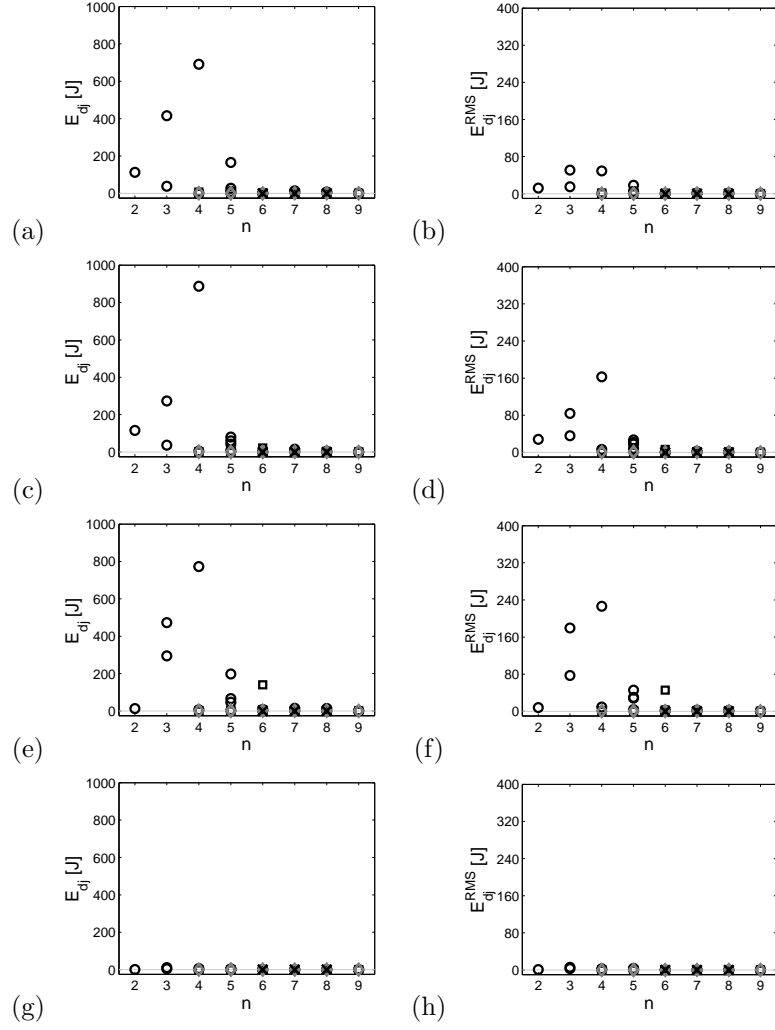


Figure 19: (left) Mean value and (right) RMS value of the modal contributions to the deformation energy  $E_{dj}(t)$  in the one-way simulation of the silo group. The first row shows silo 1, the second row silo 8, the third row silo 33 and the last row silo 40. The deformation energy for the lowest 50 eigenmodes is plotted as a function of  $n$  while separate mode shapes  $(1, n)$  are depicted as a circle ( $\circ$ ),  $(2, n)$  as a square ( $\square$ ),  $(3, n)$  as a diamond ( $\diamond$ ), and  $(4, n)$  as a cross ( $\times$ ).

the drag coefficients for these silos in figure 9. It is interesting that also at these locations eigenmodes  $(1, 3)$  and  $(1, 4)$  are predominantly excited, both statically

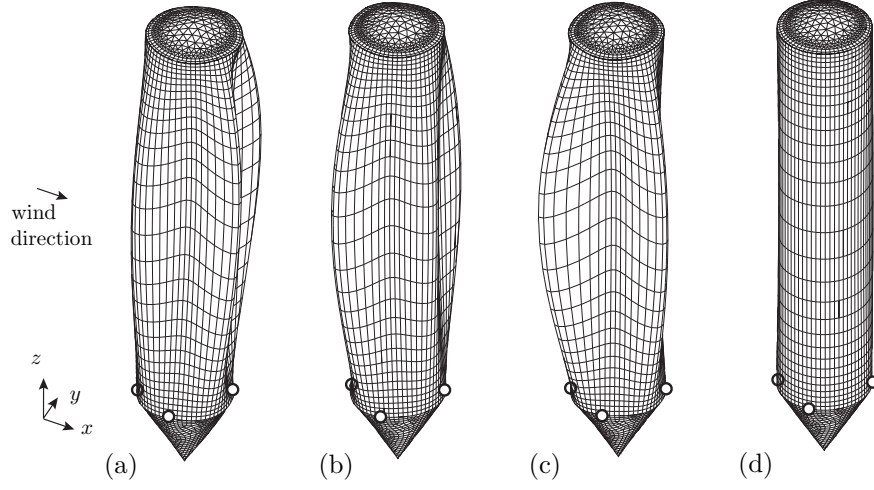


Figure 20: Displacement at  $t = 10$  s of (a) silo 1, (b) silo 8, (c) silo 33, and (d) silo 40 of the one-way simulation of the silo group. The structural displacements are mainly static and are amplified with a scale factor of 40.

and dynamically. This is related to the fact that these modes have the lowest eigenfrequencies and are hence most susceptible to the low frequency content of the turbulent wind excitation. Furthermore, the vicinity of the neighbouring silos in a square arrangement and the resulting high wind velocities that are developed in the gaps between two adjacent silos automatically gives rise to the excitation of mode (1, 4). Despite the differences in the RMS values of the modal contributions to the deformation energy between silos 8 and 33 on the one hand and silo 1 on the other hand, it should be noted that the displacements found on corner silos 8 and 33 are of the same order of magnitude as for silo 1 at the windward side, as illustrated in figure 20, mainly due to the dominance of the mean value in the deformation energy.

Finally, the results of the one-way simulation for the lee side corner silo 40 are shown in figures 19g and h. As opposed to the other corner silos, the modal deformation energy for this silo is negligibly small for all eigenmodes. The remarkable difference between the structural response at the lee side corner

661 (silo 40) and the other three corner silos (silos 1, 8 and 33) of the group is  
662 visualized in figure 20. The radial static displacements are only approximately  
663 0.01 m while vibration amplitudes are negligibly small for silo 40.

664 Overall, the patterns of the predicted vibration correspond well with the  
665 observed ovalling vibrations in the Antwerp silo group during the 2002 storm.  
666 Vibrations are observed at the windward side of the group and the modes that  
667 are preferentially excited seem to be those giving rise to the observed ovalling  
668 motion. In these one-way simulations, however, the magnitude of the vibration  
669 levels (approx. 0.01 to 0.02 m) is still much smaller than the ones observed  
670 (approx. 0.1 m) in October 2002.

#### 671 *Two-way simulations*

672 Although the one-way simulations indicated which silos were subjected to  
673 ovalling vibrations, the resulting vibration amplitudes are smaller than observed.  
674 Therefore, a fully coupled WSI simulation is performed for the group configu-  
675 ration, taking into account aeroelastic effects and/or interactions between the  
676 wake-induced excitation and the vibration. In order to obtain results within a  
677 reasonable computation time, only a single flexible silo at the windward side of  
678 the group (silo 1) is considered while all other silos in the group are rigid.

679 At the start of the two-way simulation, a direct transfer of the large struc-  
680 tural displacements to the flow field inevitably leads to a long and significant  
681 transition in the fluid and structural response. The corresponding amplitudes  
682 of the vibrations are so large that the flexible silo collides with its neighbour-  
683 ing rigid silos due to the narrow gaps. To solve this issue, the aerodynamic  
684 forces that are passed from the flow solver to the structural solver are gradually  
685 increased during a limited period of time (5 s).

686 In general, approximately 5 IQN-ILS coupling iterations are required per  
687 time step. As a result, simulating 1 s of wind flow in this two-way simulation of  
688 the silo group requires about 70 hours of computing time.

689 Figure 21 depicts the modal deformation energy as a function of time for  
690 the two-way simulation of silo 1. The initial part of the simulation until  $t = 5$  s

691 where the solution gradually evolves to the stationary regime should obviously  
 692 be disregarded but from  $t = 5$  s on the response of the structure to the full  
 693 aerodynamic forces is obtained. The fluctuations in the deformation energy  
 694 reach far beyond the corresponding levels in any of the one-way simulations. At  
 695 approximately  $t = 8$  s a peak is reached and the structural response decreases  
 696 before rising again at approximately  $t = 12$  s. This two-way simulation shows  
 697 that WSI simulations are recommended to investigate ovalling vibrations in silo  
 698 groups.

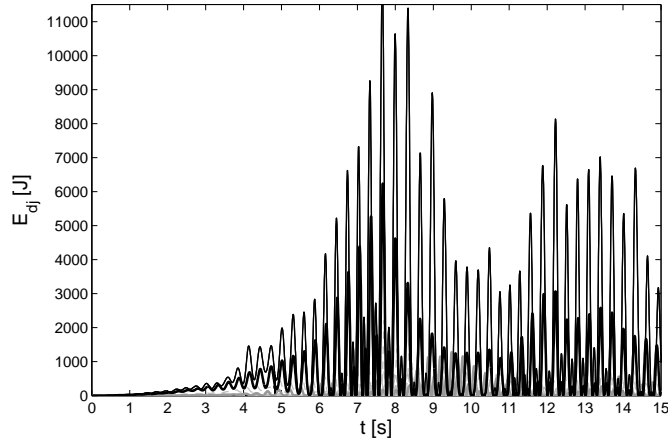


Figure 21: Modal deformation energy  $E_{dj}(t)$  for the first 20 mode shapes, based on the structural response of the upwind silo 1 in the two-way simulation of the entire silo group:  $\Phi_1 = (1, 3)$  (bold grey line),  $\Phi_2 = (1, 3)$  (bold black line),  $\Phi_4 = (1, 4)$  (thin black line), and the remaining mode shapes  $\Phi_j$  (dashed grey lines, with small energy content).

699 Taking into account the difference in vertical scale in figures 22a and b and  
 700 19a and b, it is observed that the result of the two-way simulation of silo 1  
 701 (figures 22a and b) is distinctly different from the one in the one-way simulation  
 702 (figures 19a and b). As opposed to the one-way simulations, the mean value and  
 703 the RMS value of the modal contributions to the deformation energy have the  
 704 same order of magnitude in the two-way simulations. The modal deformation  
 705 energy graphs indicate a marked aeroelastic effect and/or interaction between  
 706 the wake-induced excitation and the vibration in the structural response as a

707 result of changing wind flow patterns in vicinity of the silo structure. To distin-  
 708 guish an aeroelastic effect from interaction between the wake-induced excitation  
 709 and the vibration, the calculations would have to be repeated for higher wind  
 710 flow velocities. Aeroelastic effects would have to remain present above the crit-  
 711 ical velocity, whereas interaction between the wake-induced excitation and the  
 712 vibration is expected to disappear when the wind velocity exceeds the resonant  
 713 peak. However, repeating the analysis for different wind speeds is beyond the  
 714 scope of this work. Hence, no statements on aeroelastic effect versus interaction  
 715 between the wake-induced excitation and the vibration can be made based on  
 716 the present study. In a study by Païdoussis et al. (1988) ovaling was identified  
 717 as an aeroelastic flutter phenomenon using a model with two-dimensional flow  
 718 around a single shell that is supported at both ends. As the silo group is only  
 719 fixed at one end and the flow around it has been shown to be three-dimensional,  
 720 it is not certain whether this conclusion by Païdoussis et al. (1988) holds for  
 721 this silo group as well.

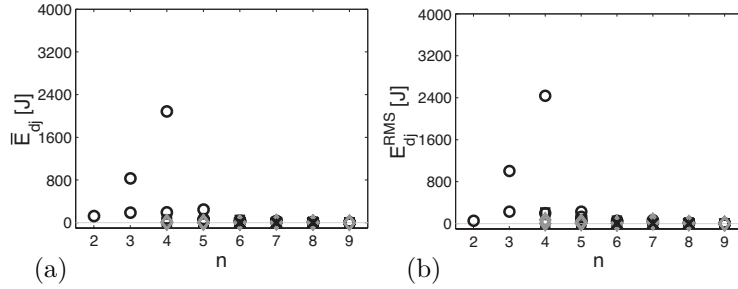


Figure 22: (a) Mean value and (b) RMS value of the modal contributions to the deformation energy  $E_{dj}(t)$  for silo 1 in the two-way simulation of the silo group. The deformation energy for the lowest 50 eigenmodes is plotted as a function of  $n$  while separate mode shapes  $(1, n)$  are depicted as a circle ( $\circ$ ),  $(2, n)$  as a square ( $\square$ ),  $(3, n)$  as a diamond ( $\diamond$ ), and  $(4, n)$  as a cross ( $\times$ ).

722 To assess the magnitude of the structural vibrations in the two-way simula-  
 723 tion, the time history of the radial displacements at two discrete points (A and  
 724 B in figure 23) along the circumference and at mid-height of silo 1 is shown in



725 figure 24 with positive values indicating outward displacements. In the small  
 726 gap between silos 1 and 9 (point A, figure 24a), the vibration amplitudes in-  
 727 crease up to about 0.05 m. At the lee side of the silo (point B, figure 24b) the  
 728 increase in the amplitude is slightly smaller but still very distinct. The vibra-  
 729 tions in this simulation are significantly larger than in the one-way simulations  
 730 and tend to the observed vibration amplitudes during the storm in 2002 (order  
 731 of magnitude of 0.1 m). Furthermore, the dominant excitation of eigenmodes  
 732 (1,3) and (1,4) is in line with observations.

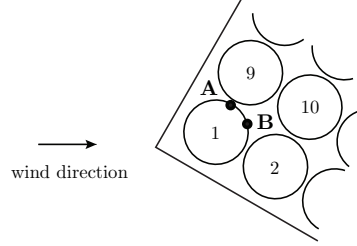


Figure 23: Location of two discrete points A and B along the circumference and at mid-height ( $z_s = 12.5$  m) of silo 1 in the group arrangement.

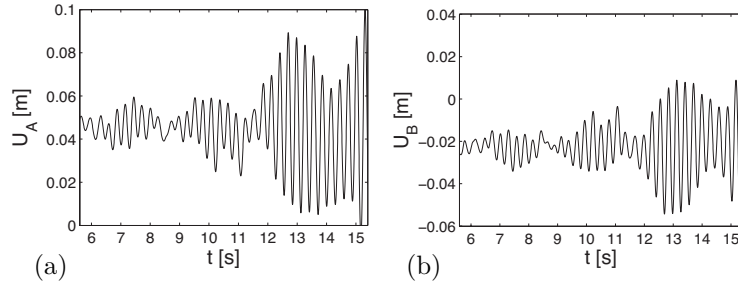


Figure 24: Radial displacements at two discrete points A and B (cfr. figure 23) along the circumference and at mid-height of silo 1 in the two-way simulation of the group arrangement.

## 733 6. Conclusions

734 The comparison between the one-way and two-way simulations did not re-  
735 veal significant differences for the single silo case. By contrast, the vibration  
736 amplitudes for the silo group are significantly larger in the two-way simulations  
737 than in the one-way simulations. As the importance of two-way simulations  
738 is difficult to estimate a priori, it is thus recommended to take the structural  
739 displacement into account in the wind flow simulation for WSI simulations of  
740 similar structures.

741 It is also concluded that the average and/or the RMS value of the wind load-  
742 ing on the windward side of a silo group can be much larger than on a single  
743 silo. This observed group effect is in agreement with the outcome of the Ferry-  
744 bridge investigation, but appears on the windward side of closely spaced silos  
745 as opposed to the leeward side of more widely spaced cooling towers. Further-  
746 more, comparing the two-way simulations for the single silo with the two-way  
747 simulations for the silo group demonstrates that the vibration amplitudes are  
748 significantly larger for the group. So, both the Ferrybridge investigation and  
749 this study indicate that objects in group should not be designed using wind  
750 loading data or norms for a single object.

751 Finally, the two-way simulations of the silo group show results in agreement  
752 with the observations during the storm. Extensive analysis of the grid size and  
753 time step size gives an idea of the uncertainty on the results. However, the  
754 duration of the simulations mentioned throughout this article and the required  
755 computing power limit the use of the proposed modelling as engineering tool to  
756 special investigations at present.

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